

Fin Efficiency Design of Micro-Channel for Nd: YAG Slab Laser

Rajesh Pant, Jasmeet Kalra, Jagdish Singh Mehta, Pankaj Negi

Abstract: The heat generated by a Slab Lasers can exceed 1,000 watts but the area available for cooling is too small. This results in localization of heat flux which makes heat dissipation a challenge in slab lasers. The Heat transfer coefficient can increase up to a very high range, which can't be efficiently achieved by the conventional water cooling. Micro-channel coolers address this problem competently. These channels contain liquid, which transfers heat to the sink with high efficiency. The objective of this paper is to design the micro-channel coolers that will be efficiently capable of removing the heat from the slab laser without causing any thermal distortion in the laser. The dimension of the slab is given as follows:-Length(l)=50mm, Width(w)=8mm and thickness(t)=2mm.

Key Words: MCC, Micro-channel, slab laser, heat sink.

I. INTRODUCTION

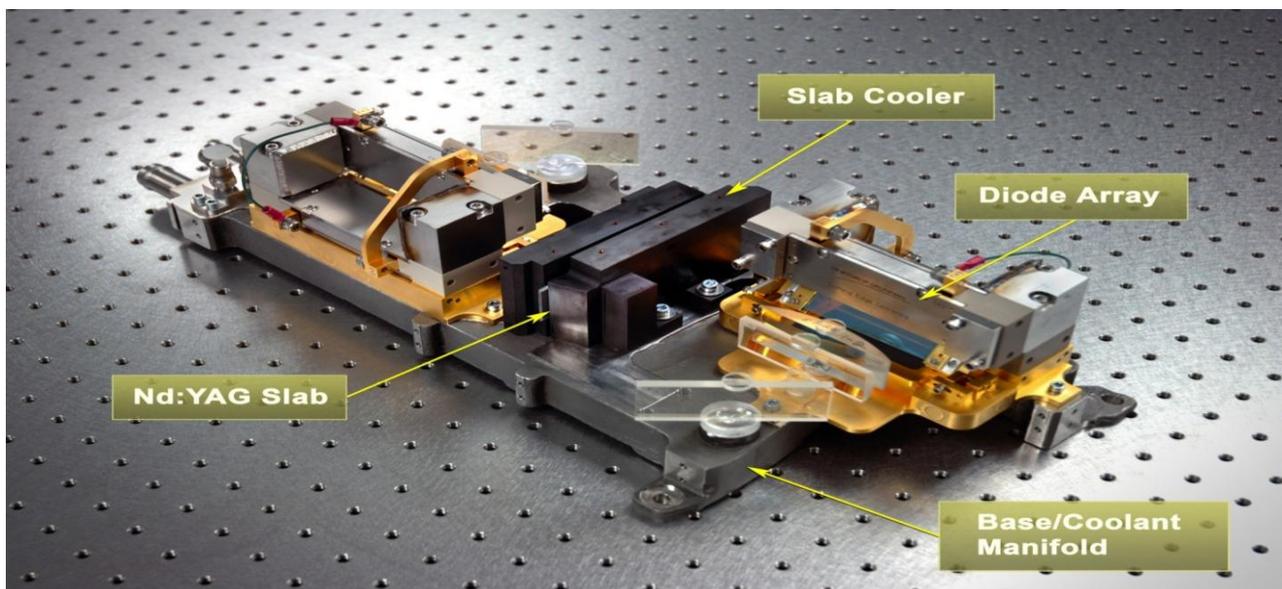


Figure 1 Nd:YAG laser with lid open and with microchannel cooling setup

II. Nd: YAG SLAB LASERS

Nd: YAG Slab lasers falls in the category of solid lasers which uses slab form crystal as gain medium. A slab laser has a very less thickness in comparison to other two dimensions. However, some lasers are included in category of slab laser not because they have large aspect ratio but due to their rectangular geometry.

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These lasers use diodes or flash lamps as pumping source. The wavelength of light emitted by Nd:YAG lasers is of 1064 nm.

YAG crystal properties

- Formula: $Y_3Al_5O_{12}$
- Crystal Structure: Cubic
- Hardness(Moh): 8–8.5
- Melting point: 1970 degree centigrade

Properties of Nd:YAG at 25°C (with 1% Nd doping)

- Formula: $Y_{2.97}Nd_{0.03}Al_5O_{12}$
- Thermal conductivity: $0.14 W \cdot cm^{-1} \cdot K^{-1}$
- Specific heat capacity: $0.59 J \cdot g^{-1} \cdot K^{-1}$
- Thermal expansion: $6.9 \times 10^{-6} K^{-1}$
- Poisson's ratio: 0.3

III. PROBLEM DEFINITION

A Nd:YAG slab Laser is capable of generating heat of 900W. The objective is to design the micro-channel coolers that will be efficiently capable of removing the heat from the slab laser without causing any thermal distortion in the laser. The dimension of the slab is given as follows:- Length(l) =50mm, Width(w) =8mm and thickness(t)=2mm. The micro-channels will be placed in contact with the two surfaces of the lasers for the cooling and the other two sides will remain un-cooled.

Assumptions:

1. There is an only one dimensional heat flow in the slab i.e. In the Y-direction or along the thickness and the other two dimension or sides remain isolated.
2. The Optical axis and the Z-axis of the slab coincide.

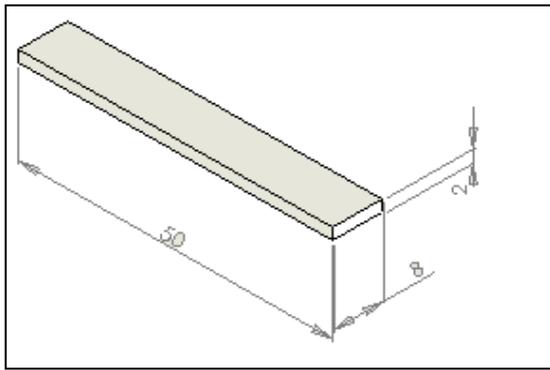


Figure 2 Nd: YAG Slab Geometry

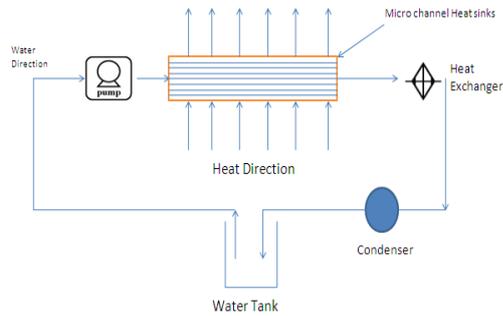


Fig. 3 A simple Diagrammatic representation of Micro-channel

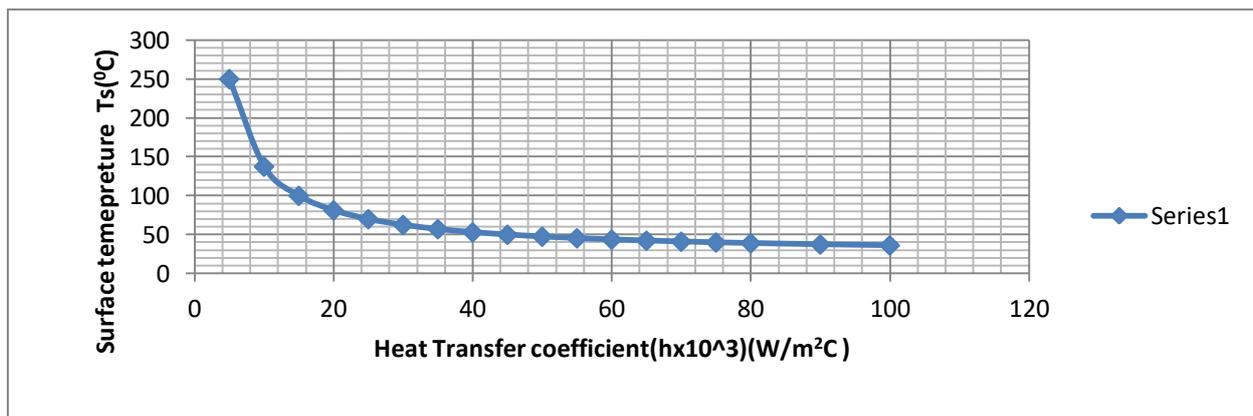


Figure 4 Surface temp. vs heat Transfer coefficient plot

From the above plot let us select the heat transfer coefficient of $30 \times 10^3 \text{ W/m}^2\text{K}$ as our operating value and temperature

corresponding to the heat transfer coefficient is 62.5°C .

IV. THERMAL ANALYSIS

The Thermal analysis is done only in the y-direction .the temperatures at different section are given by equation (1).

$$T_c - T_s = \frac{t^2}{8k} Q$$

Where T_c =Center temperature of Slab, T_s =Surface temperature of Slab, t = thickness of the slab, K =Thermal conductivity of Slab, Q =Heat load of slab

$$T_c - T_s = \frac{.002^2}{8 \times 14 \times .008 \times .002 \times .05} * 900$$

$$T_c - T_s = 40.17^\circ\text{C}$$

(1)

assuming the uniform distribution of heat from the two upper surfaces. The amount of heat required to be dissipated by the water from one surface is 450W.

$$Q = h \times A \times \Delta T$$

(2)(From one surface)

Where h =heat transfer coefficient, A =area of slab in contact with water, $\Delta T = T_s - T_f$ and T_f =Fluid or Water temperature= 25°C

$$450 = h \times 0.008 \times 0.05 \times (T_s - 25)$$

(2)

Now we have to calculate the value of surface temperature and the heat transfer coefficient. For this we will plot a curve between heat transfer coefficient and the surface temperature and will calculate the surface temperature by varying the heat transfer coefficient.

Hence the value of T_s equals to 62.5°C .
Now substituting the values in equation (1) we get
 $T_c=102.67^\circ\text{C}$.

Now we will check our results with the help of the ANSYS and will perform its thermal analysis.

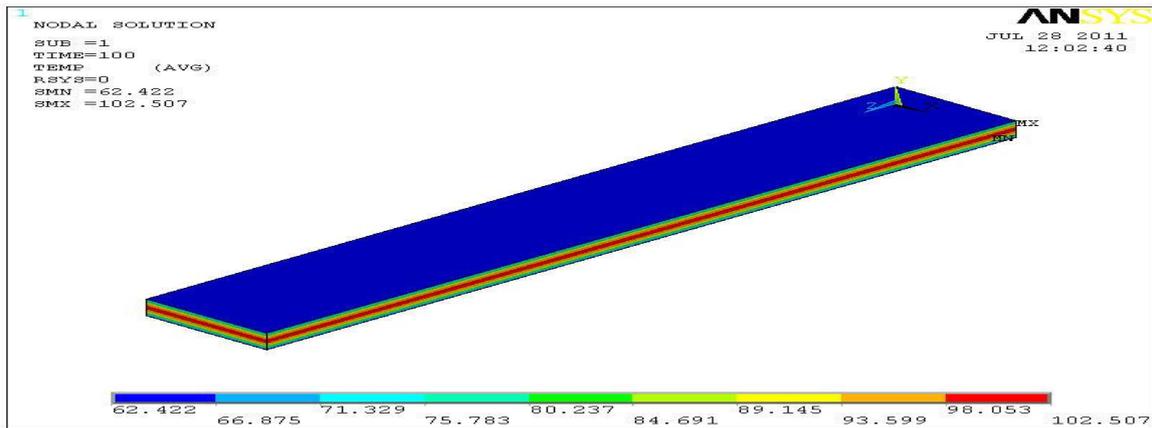


Figure 5 Thermal Analysis of Slab with ANSYS

Modeling Details of the analysis:-

1. Element type - Thermal Mass-Solid brick
8node70

2. Hex meshing with sweep
3. Full transient analysis.

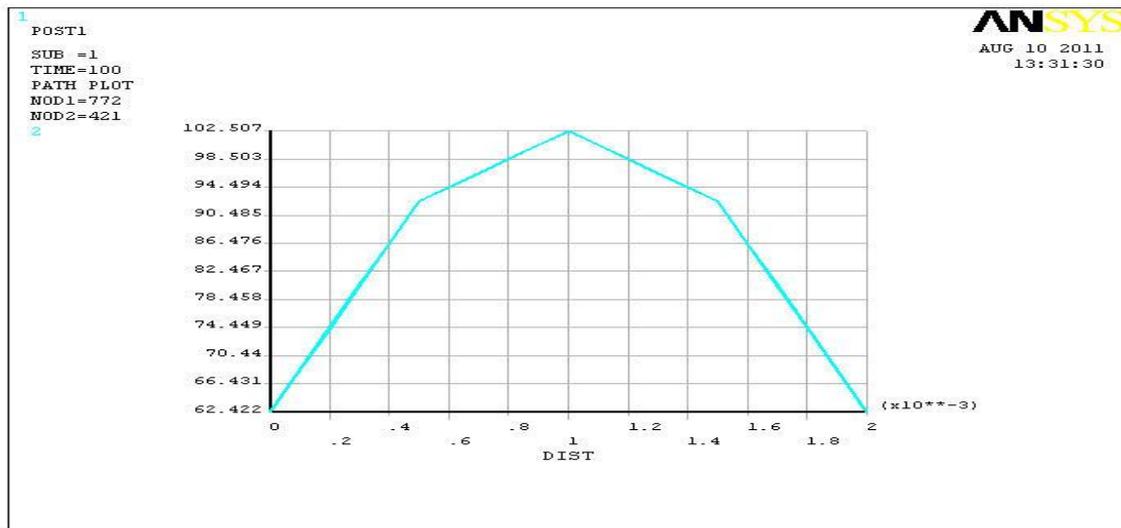


Figure 6 ANSYS Generated Temperature distribution curve in Laser Slab

As we can see that our mathematical results of center temperature of the micro-channel slab agrees with the ANSYS generated temperature distribution curve.

V. CALCULATION OF FIN EFFICIENCY

The fin efficiency of micro-channels is:-

$$\eta = \frac{\tanh N}{N} \quad (3)$$

Where

$$N = (2h/k_w W_w)^{1/2} z \quad (4)$$

h =heat transfer coefficient= $31200\text{W/m}^2\text{C}$, Thermal conductivity (K_w)= 398W/mC , Fin width(W_w)= $100\mu\text{m}$, channel depth(z)= $600\mu\text{m}$.

Putting all the values in the equation (4):-

$$N = \left(\frac{2 \times 31200}{398 \times 100 \times 10^{-06}} \right)^{0.5} * 600 * 10^{-06}$$

$$N = 0.781.$$

Substituting the value of N in the equation (3):-

$$\eta = \frac{\tanh(0.781)}{0.781}$$

$$\eta = \frac{0.653}{0.781} = 0.8361$$

Hence the fin efficiency of the channel comes to be **83.61%**

VI. RESULTS AND DISCUSSION

Following are the results obtained from the calculation of the micro-channel heat sinks:

A micro-channel having the configuration Length(l) = 50mm ,

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Width (w) = 8mm, thickness (t) = 2mm, h = heat transfer coefficient = 31200 W/m²C, Thermal conductivity (K_w) = 398 W/mC, Fin width (W_w) = 100 μ m and channel depth (z) = 600 μ m can easily dissipate heat up to 900W with an efficiency of 83.61% which is far better than the conventional method. This problem has been solved mathematically and our results agree with the ANSYS generated curve of temperature distribution.

The above study and calculation of Micro-channel coolers proved that the micro-channels are the advance and the better way of cooling the lasers than the other conventional methods. From the above study it can also be concluded that

1) In micro-channel heat sink design the convective heat transfer coefficient play significant role and conductive heat transfer coefficient has negligible effect on heat dissipation. Hence the main focus should be on internal design of micro channel for better performance.

2) Fluid flow and thermal performance can be optimized by improving the geometry of micro channel design and finding balance between pressure drop and heat transfer constraints.

Result obtained helped in concluding that the temperature generated in the thin slab lasers is quite high and needs to be removed by an efficient cooling method, it has also been seen that the heat spreaders having high conductivity proves to be a better. Lastly these results proved that the micro-channels are smartly able to dissipate the heat generated inside the lasers through convection.

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